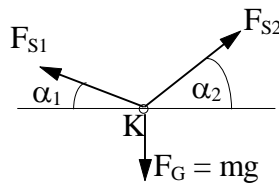


## Lösungen für Aufgaben zur Technischen Mechanik – Statik -

### Lösung 2.1.8



$$\rightarrow: -F_{S1}\cos\alpha_1 + F_{S2}\cos\alpha_2 = 0 \quad F_{S2} = \frac{F_{S1}\cos\alpha_1}{\cos\alpha_2}$$

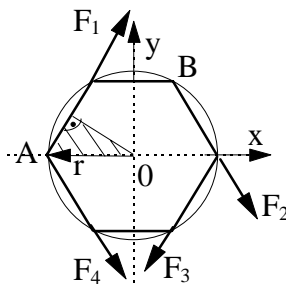
$$\uparrow: F_{S1}\sin\alpha_1 + F_{S2}\sin\alpha_2 - mg = 0 \quad F_{S1}(\sin\alpha_1 + \tan\alpha_2 \cos\alpha_1) = mg$$

$$\cos\alpha_1 = \frac{b}{l} = \frac{4}{4,5} = 0,8889 \quad \alpha_1 = 27,3^\circ$$

$$\tan\alpha_2 = \frac{a + \sqrt{l^2 - b^2}}{c - b} = \frac{4,062}{6} = 0,677 \quad \alpha_2 = 34,1^\circ$$

Ergebnis:  $F_{S1} = 277\text{N}$ ,  $F_{S2} = 297\text{N}$

### Lösung 2.2.3



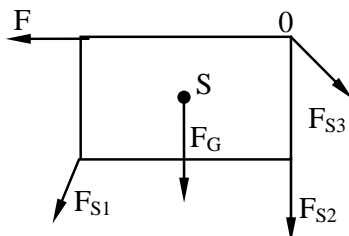
$$F_{Rx} = \sum F_{ix} = F_1\cos 60^\circ + F_2\cos 60^\circ - F_3\cos 60^\circ + F_4\cos 60^\circ = 2F\cos 60^\circ = F$$

$$F_{Ry} = \sum F_{iy} = F_1\sin 60^\circ - F_2\sin 60^\circ - F_3\sin 60^\circ - F_4\sin 60^\circ = -2F\sin 60^\circ = -F\sqrt{3}$$

$$F_R = \sqrt{F_{Rx}^2 + F_{Ry}^2} = \sqrt{F^2 + 3F^2} = 2F$$

$$\tan\alpha_R = \frac{F_{Ry}}{F_{Rx}} = -\tan 60^\circ \quad \alpha_R = 300^\circ$$

### Lösung 2.2.7



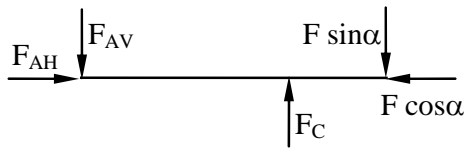
$$\leftarrow: F + F_{S1} \cos 60^\circ - F_{S3} \cos 45^\circ = 0$$

$$\uparrow: -F_{S2} - F_{S1} \sin 60^\circ - F_{S3} \sin 45^\circ - F_G = 0$$

$$\curvearrow 0: 2F_{S1}a \sin 60^\circ - F_{S1}a \cos 60^\circ + F_G a = 0$$

Ergebnisse:  $F_{S1} = -812\text{ N}$ ....Druckstab  
 $F_{S2} = -891\text{ N}$ ....Druckstab  
 $F_{S3} = 840\text{ N}$ ....Zugstab

### Lösung 3.1.2



$$\rightarrow : F_{AH} - F \cos \alpha = 0$$

$$\uparrow : -F_{AV} - F \sin \alpha + F_C = 0$$

$$\curvearrow A : -F \sin \alpha \cdot 3a + F_C \cdot 2a = 0$$

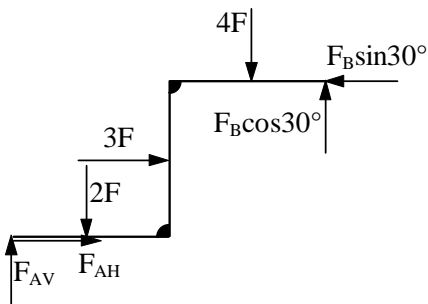
Ergebnis:

$$F_C = \frac{3}{2} F \sin \alpha = \frac{3}{4} \sqrt{3} F = 39 \text{ kN}$$

$$F_{AH} = F \cos \alpha = \frac{1}{2} F = 15 \text{ kN}$$

$$F_{AV} = \frac{1}{2} F \sin \alpha = \frac{\sqrt{3}}{4} F = 13 \text{ kN}$$

### Lösung 3.1.12



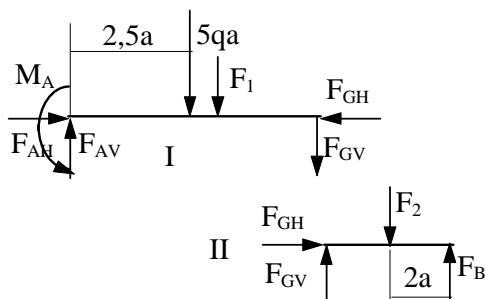
$$\rightarrow : F_{AH} + 3F - F_B \sin 30^\circ = 0$$

$$\uparrow : F_{AV} + F_B \cos 30^\circ - 2F - 4F = 0$$

$$\curvearrow A : F_B \cos 30^\circ \cdot 4a + F_B \sin 30^\circ \cdot 2a - 2F \cdot a - 3F \cdot a - 4F \cdot 3a = 0$$

Ergebnis:  $F_B = 3,82F$ ;  $F_{AH} = -1,09F$ ;  $F_{AV} = 2,7F$

### Lösung 3.2.3



Teil I:

$$\rightarrow : F_{AH} - F_{GH} = 0$$

$$\uparrow : F_{AV} - F_{GV} - F_1 - 5qa = 0$$

$$\curvearrow A : M_A - F_1 \cdot 3a - 5qa \cdot 2,5a - F_{GV} \cdot 5a = 0$$

Teil II:

$$\rightarrow : F_{GH} = 0$$

$$\uparrow : F_B + F_{GV} - F_2 = 0$$

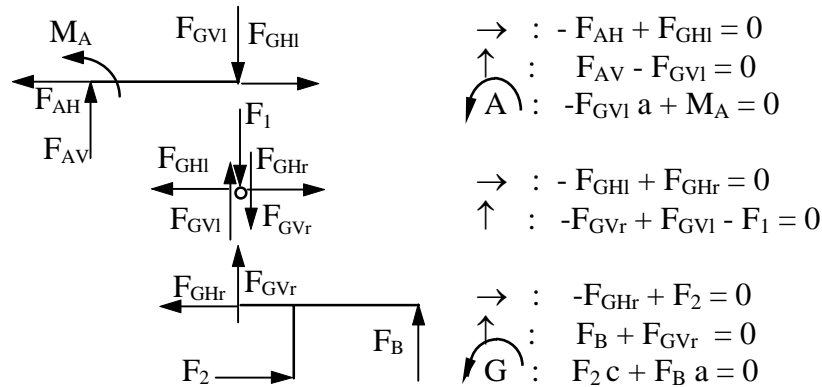
$$\curvearrow G : -F_2 \cdot 3a + F_B \cdot 5a = 0$$

Ergebnis:

$$F_B = \frac{3}{5}F_2 = 960\text{N} \quad F_{GH} = 0 \quad F_{GV} = \frac{2}{5}F_2 = 640\text{N} \quad F_{AH} = 0 \quad F_{AV} = F_1 + \frac{2}{5}F_2 + 5qa = 2440\text{N}$$

$$M_A = 3F_1a + 2F_2a + \frac{25}{2}qa^2 = 8100\text{Nm}$$

### Lösung 3.2.9



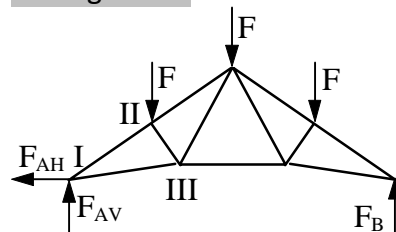
Ergebnis:

$$F_{AH} = F_2; \quad F_{GHl} = F_{GHr} = F_2; \quad F_B = -F_2 \cdot \frac{c}{a}$$

$$F_{AV} = F_1 + F_2 \cdot \frac{c}{a}; \quad F_{GVl} = F_1 + F_2 \cdot \frac{c}{a};$$

$$M_A = F_1 \cdot a + F_2 \cdot c; \quad F_{GVr} = F_2 \cdot \frac{c}{a}$$

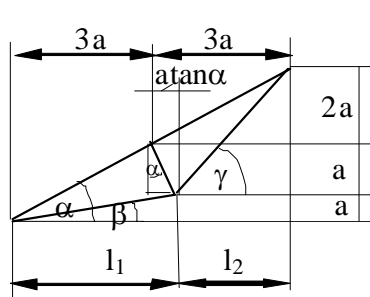
### Lösung 3.2.22



Symmetrie:

$$F_{AH} = 0; \quad F_{AV} = F_B = 1,5F = 30\text{kN}$$

Zur Geometrie:

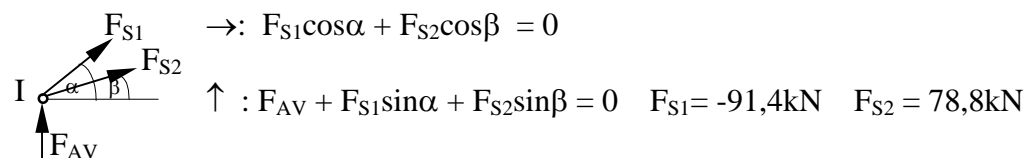


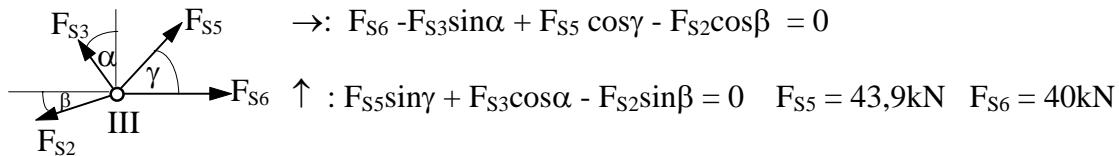
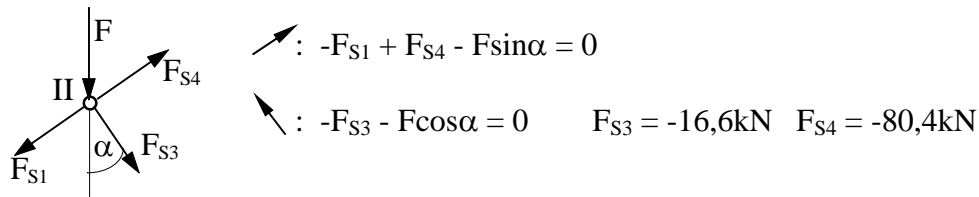
$$l_1 = 3a + a \tan \alpha; \quad \tan \alpha = \frac{4a}{6a} = \frac{2}{3}; \quad l_1 = \frac{11}{3}a$$

$$l_2 = 6a - l_1 = \frac{7}{3}a$$

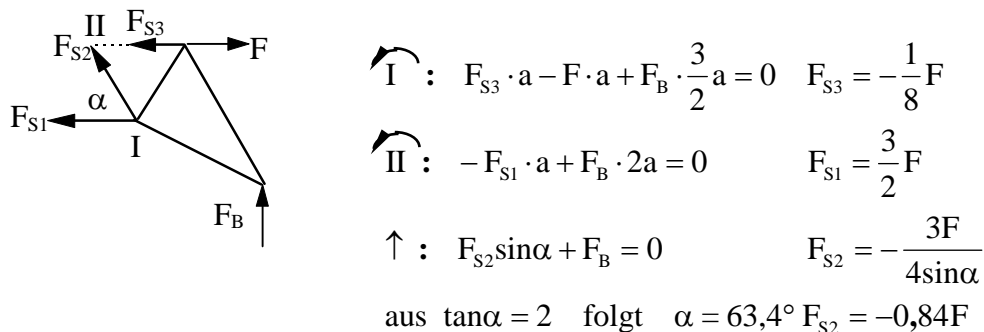
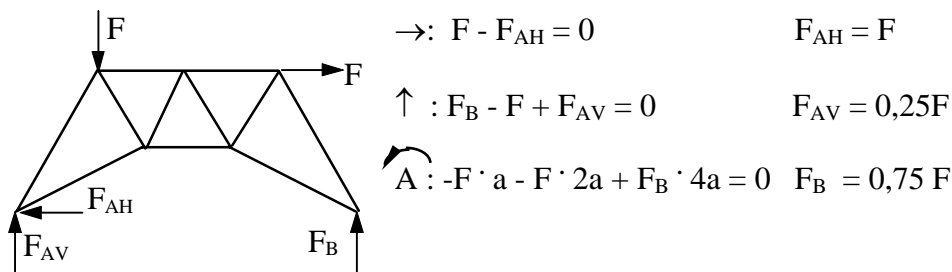
$$\tan \alpha = \frac{2}{3} \rightarrow \alpha = 33,7^\circ \quad \tan \beta = \frac{a}{l_1} = \frac{3}{11} \rightarrow \beta = 15,25^\circ$$

$$\tan \gamma = \frac{3a}{l_2} = \frac{9}{7} \rightarrow \gamma = 52,15^\circ$$

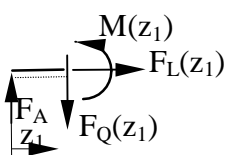
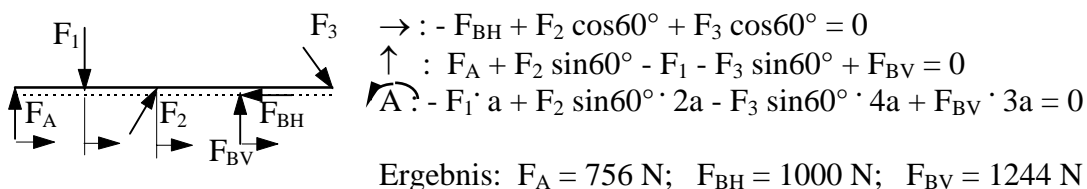




### Lösung 3.2.25

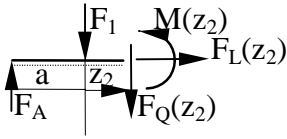


### Lösung 3.3.2



$0 \leq z_1 \leq a$   
 $\rightarrow: F_L(z_1) = 0; \quad \uparrow: F_Q(z_1) = F_A = 756 \text{ N}$   
 $\curvearrowright_{S1}: M(z_1) = F_A z_1;$   
 $z_1 = 0: M(0) = 0 \quad z_1 = a: M(a) = 756 \text{ Nm}$

$$0 \leq z_2 \leq a$$

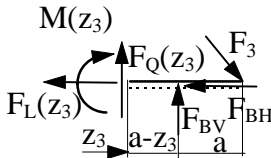


$$\rightarrow : F_L(z_2) = 0; \quad \uparrow : F_Q(z_2) = F_A - F_1 = -1244 \text{ N}$$

$$\curvearrow S_2 : M(z_2) = F_A (a + z_2) - F_1 z_2;$$

$$z_2 = 0: M_2(0) = 756 \text{ Nm} \quad z_2 = a: M_2(a) = -488 \text{ Nm}$$

$$0 \leq z_3 \leq a$$



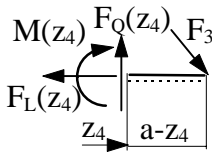
$$\rightarrow : F_L(z_3) = -F_{BH} + F_3 \cos 60^\circ = -500 \text{ N};$$

$$\uparrow : F_Q(z_3) = -F_{BV} + F_3 \sin 60^\circ = -378 \text{ N};$$

$$\curvearrow S_3 : M(z_3) = F_{BV} (a - z_3) - F_3 \sin 60^\circ \cdot (2a - z_3);$$

$$z_3 = 0: M_3(0) = -488 \text{ Nm} \quad z_3 = a: M_3(a) = -866 \text{ Nm}$$

$$0 \leq z_4 \leq a$$

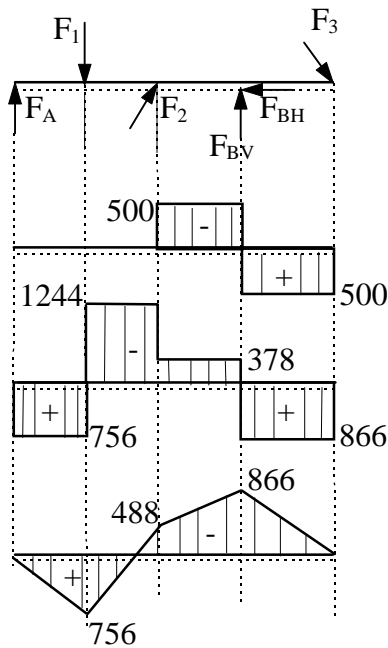


$$\rightarrow : F_L(z_4) = F_3 \cos 60^\circ = 500 \text{ N};$$

$$\uparrow : F_Q(z_4) = F_3 \sin 60^\circ = 866 \text{ N};$$

$$\curvearrow S_4 : M(z_4) = -F_3 \sin 60^\circ \cdot (a - z_4);$$

$$z_4 = 0: M_4(0) = -866 \text{ Nm} \quad z_4 = a: M_4(a) = 0$$



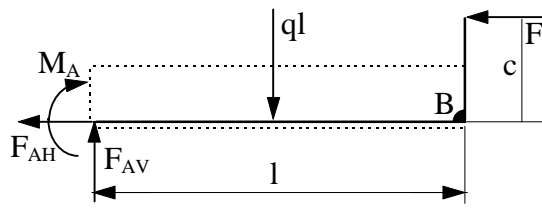
$F_L$  in N

$F_Q$  in N

M in Nm

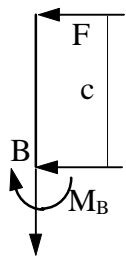
$$|M_{\max}| = 866 \text{ Nm}$$

### Lösung 3.3.6



$$\overset{\curvearrowright}{A}: M_A = Fc - \frac{1}{2}ql^2 = ql\left(c - \frac{l}{2}\right)$$

$$M_A \text{ ist } \begin{cases} \text{positiv} & \text{für } c > \frac{l}{2} \\ 0 & \text{für } c = \frac{l}{2} \\ \text{negativ} & \text{für } c < \frac{l}{2} \end{cases}$$

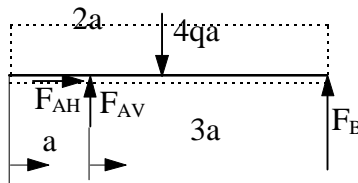


$$\overset{\curvearrowright}{B}: M_B = Fc = qlc$$

$M_B$  ist positiv für beliebige  $c$ , bei  $c \geq \frac{l}{2}$  ist  $M_B > M_A$ ,  $|M_A| = |M_B|$  ist nur für negatives  $M_A$  möglich.

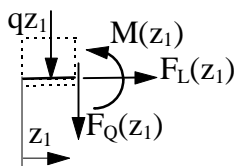
$$-M_A = M_B \text{ oder } -ql\left(c - \frac{l}{2}\right) = qlc \quad c = \frac{l}{4}$$

### Lösung 3.3.7



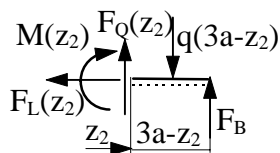
$$\begin{aligned} \rightarrow: F_{AH} &= 0 \\ \uparrow: -4qa + F_{AV} + F_B &= 0 \\ \overset{\curvearrowright}{A}: -4qa \cdot a + F_B \cdot 3a &= 0 \end{aligned}$$

Ergebnis:  $F_{AH} = 0$ ;  $F_{AV} = \frac{8}{3}qa = 26,7\text{kN}$ ;  $F_B = \frac{4}{3}qa = 13,3\text{kN}$ ;



$$\begin{aligned} 0 \leq z_1 \leq a \\ \rightarrow: F_L(z_1) &= 0 \\ \uparrow: F_Q(z_1) &= -qz_1 \\ z_1 = 0: F_Q(0) &= 0 \quad z_1 = a: F_Q(a) = -qa = -10\text{kN} \\ \overset{\curvearrowright}{S_1}: M(z_1) &= -\frac{1}{2}qz_1^2; \end{aligned}$$

$$z_1 = 0: M(0) = 0 \quad z_1 = a: M(a) = -0,5qa^2 = -5\text{kNm}$$



$$\begin{aligned} 0 \leq z_2 \leq 3a \\ \rightarrow: F_L(z_2) &= 0 \\ \uparrow: F_Q(z_2) &= -F_B + q(3a - z_2) \\ z_2 = 0: F_Q(0) &= \frac{5}{3}qa = 16,7\text{kN}; \end{aligned}$$

$$z_2 = 3a: F_Q(a) = -\frac{4}{3}qa = -13,3\text{kN}$$

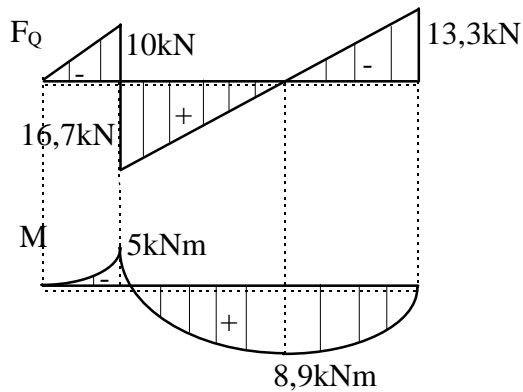
$$\overset{\curvearrowright}{S_2}: M(z_2) = F_B \cdot (3a - z_2) - \frac{1}{2}q(3a - z_2)^2;$$

$$z_2 = 0: M_2(0) = -\frac{1}{2}qa^2 = -5\text{kNm} \quad z_2 = 3a: M_2(3a) = 0$$

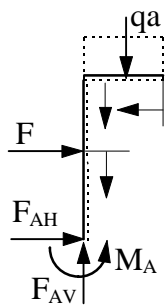
Besondere Werte:

- Querkraft-Nullstelle:  $\bar{z}_2: F_Q(\bar{z}_2) = 0 \quad \bar{z}_2 = \frac{5}{3}a = 1,66m$

- Extremwert für M:  $M(\bar{z}_2) = \frac{8}{9}qa^2 = 8,9\text{kNm} = M_{\max}$

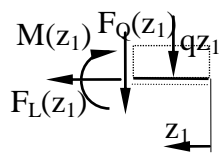


### Lösung 3.3.13

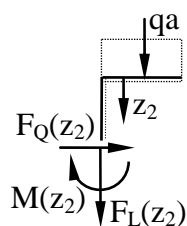


$$\begin{aligned} \rightarrow: F_{AH} + F &= 0 \\ \uparrow: F_{AV} - qa &= 0 \\ \curvearrowright_A: M_A - F \cdot a - qa \cdot 0,5a &= 0 \end{aligned}$$

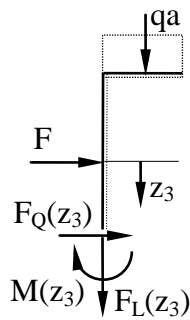
Ergebnis:  $F_{AV} = qa; F_{AH} = -qa; M_A = 1,5qa^2$



$$\begin{aligned} 0 \leq z_1 \leq a \\ \leftarrow: F_L(z_1) &= 0 \\ \uparrow: F_Q(z_1) = -qz_1 \quad F_Q(0) &= 0 \quad F_Q(a) = -qa \\ \curvearrowright_{S_1}: M(z_1) &= -0,5q z_1^2 \\ z_1 = 0: M(0) &= 0 \quad z_1 = a: M(a) = -0,5qa^2 \end{aligned}$$



$$\begin{aligned} 0 \leq z_2 \leq a \\ \leftarrow: F_Q(z_2) &= 0 \\ \downarrow: F_L(z_2) &= -qa \\ \curvearrowright_{S_2}: M(z_2) &= -0,5qa^2 \end{aligned}$$



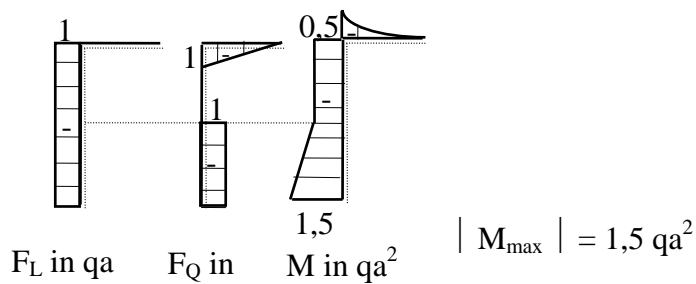
$$0 \leq z_3 \leq a$$

$$\leftarrow : F_Q(z_3) = -F = -qa$$

$$\downarrow : F_L(z_3) = -qa$$

$$\curvearrow S_2 : M(z_3) = -0,5qa^2 - Fz_3$$

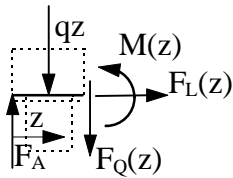
$$z_3 = 0: M(0) = -0,5qa^2 \quad z_3 = a: M(a) = -1,5qa^2$$



### Lösung 3.3.23

Aus Symmetriegründen im Teil I folgt  $F_A = F_{GV} = \frac{1}{2}qa$

$$0 \leq z \leq l$$



$$\uparrow : F_Q(z) = F_A - qz$$

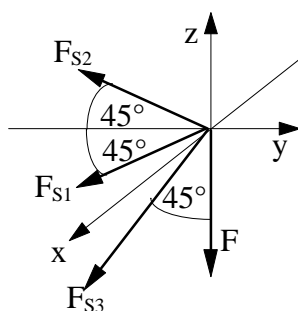
$$\curvearrow S : M(z) = F_A \cdot z - \frac{1}{2}qz^2$$

$$M(0,5a) = 0,125qa^2; \quad M(a) = 0 \quad M(l) = -0,5ql(1-a)$$

$$|M_{\max}|_I = |M_{\max}|_{II}$$

$$\frac{1}{8}qa^2 = \frac{1}{2}ql(1-a) \quad a^2 = 4l^2 - 4la \quad a^2 + 4la - 4l^2 = 0 \quad a = 2l(\sqrt{2} - 1) = 0,828l$$

### Lösung 4.1.3



$$\nearrow x: -F_{S2} \sin 45^\circ + F_{S1} \sin 45^\circ = 0 \quad F_{S1} = F_{S2}$$

$$\rightarrow y: -F_{S2} \cos 45^\circ - F_{S1} \cos 45^\circ - F_{S3} \sin 45^\circ = 0$$

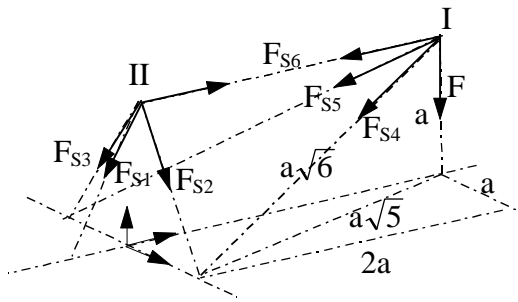
$$\uparrow z: -F - F_{S3} \cos 45^\circ = 0$$

$$\text{Ergebnis: } F_{S1} = F_{S2} = \frac{\sqrt{2}}{2}F = 707\text{N}; \quad F_{S3} = -\sqrt{2}F = -1414\text{N}$$



### Lösung 4.2.5

Aus Symmetriegründen folgt:  $F_{S4} = F_{S5}$  und  $F_{S2} = F_{S3}$



Knoten I:

$$\downarrow: F + 2F_{S5} \frac{1}{\sqrt{6}} = 0 \quad F_{S5} = F_{S4} = -\frac{\sqrt{6}}{2} F = -1,2F$$

$$\leftarrow: F_{S6} + 2F_{S5} \frac{\sqrt{5}}{\sqrt{6}} \cdot \frac{2}{\sqrt{5}} = 0 \quad F_{S6} = -\frac{2\sqrt{6}}{3} F_{S5}$$

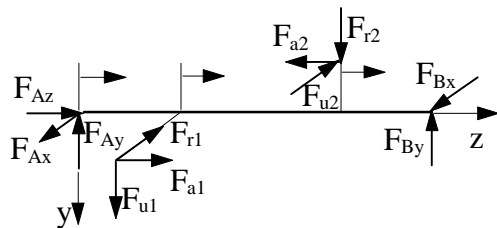
$$F_{S6} = 2F$$

Knoten II:

$$\leftarrow: -F_{S6} + F_{S1} \frac{1}{\sqrt{2}} = 0 \quad F_{S1} = \sqrt{2} F_{S6} = 2\sqrt{2} F = 2,8F$$

$$\downarrow: \frac{\sqrt{2}}{2} F_{S1} + 2 \frac{\sqrt{2}}{2} F_{S2} = 0 \quad F_{S2} = F_{S3} = -\frac{1}{2} F_{S1} = -1,4F$$

### Lösung 4.3.1



Nach dem Erstarrensprinzip gilt:

$$\rightarrow F_{u1} \cdot r_{01} - F_{u2} \cdot r_{02} = 0 \quad \text{oder} \quad F_{u2} = F_{u1} \cdot \frac{r_{01}}{r_{02}}$$

$$\begin{aligned} F_{u1} &= 4500\text{N} & F_{a1} &= 1640\text{N} & F_{r1} &= 1740\text{N} \\ F_{u2} &= 10800\text{N} & F_{a2} &= 3930\text{N} & F_{r2} &= 4180\text{N} \end{aligned}$$

Lagerreaktionen:

$$\leftarrow F_{Ax} - F_{r1} - F_{u2} + F_{Bx} = 0$$

$$\uparrow: F_{Ay} - F_{r2} - F_{u1} + F_{By} = 0$$

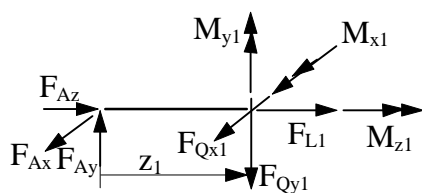
$$\rightarrow: F_{Az} + F_{a1} - F_{a2} = 0$$

$$\leftarrow A: -F_{u1}l_1 - F_{r2}(l_1 + l_2) + F_{a2}r_{02} + F_{By}(l_1 + l_2 + l_3) = 0$$

$$\downarrow A: -F_{r1}l_1 - F_{u2}(l_1 + l_2) - F_{a1}r_{01} + F_{Bx}(l_1 + l_2 + l_3) = 0$$

Ergebnis:  $F_{Ax} = 3660\text{N}$     $F_{Ay} = 5580\text{N}$     $F_{Az} = 2290\text{N}$   
 $F_{Bx} = 8880\text{N}$     $F_{By} = 3100\text{N}$

$$0 \leq z_1 \leq l_1$$



$$F_{L1} = -F_{Az} = -2290\text{N}$$

$$F_{Qx1} = -F_{Ax} = -3360\text{N}$$

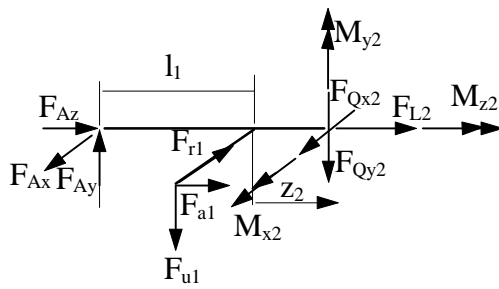
$$F_{Qy1} = F_{Ay} = 5580\text{N}$$

$$M_{x1} = F_{Ay} \cdot z_1 \quad M_{x1}(0) = 0 \quad M_{x1}(l_1) = 307\text{Nm}$$

$$M_{y1} = -F_{Ax} \cdot z_1 \quad M_{y1}(0) = 0 \quad M_{y1}(l_1) = -201\text{Nm}$$

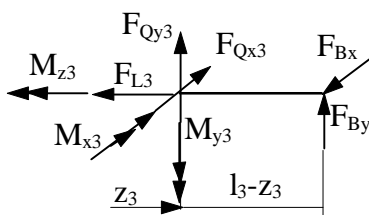
$$M_{z1} = 0$$

$$0 \leq z_2 \leq l_2$$



$$\begin{aligned}
 F_{L2} &= -F_{Az} - F_{a1} = -3930\text{N} \\
 F_{Qx2} &= -F_{Ax} + F_{r1} = -1920\text{N} \\
 F_{Qy2} &= F_{Ay} - F_{u1} = 1080\text{N} \\
 M_{x2} &= F_{Ay} \cdot (l_1 + z_2) - F_{u1} \cdot z_2 \\
 M_{x2}(0) &= 307\text{Nm} \quad M_{x2}(l_2) = 383\text{Nm} \\
 M_{y2} &= -F_{Ax} \cdot (l_1 + z_2) + F_{r1} \cdot z_2 - F_{a1} \cdot r_{01} \\
 M_{y2}(0) &= -398\text{Nm} \quad M_{y2}(l_2) = -533\text{Nm} \\
 M_{z2} &= -F_{u1} \cdot r_{01} = -540\text{Nm}
 \end{aligned}$$

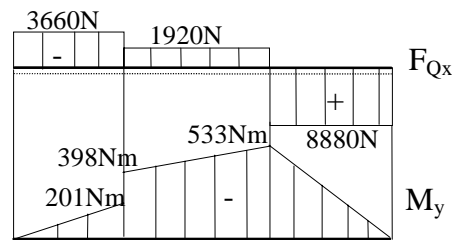
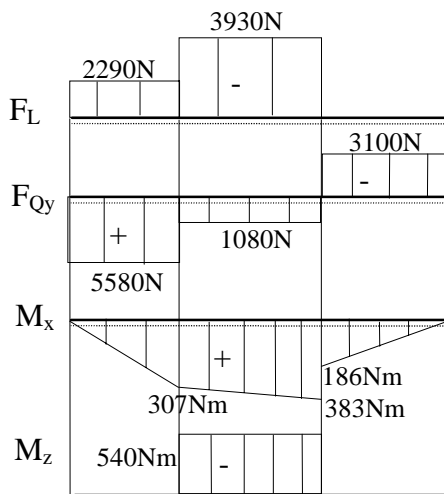
$$0 \leq z_3 \leq l_3$$



$$\begin{aligned}
 F_{L3} &= 0 \\
 F_{Qx3} &= F_{Bx} = 8880\text{N} \\
 F_{Qy3} &= -F_{By} = -3100\text{N} \\
 M_{x3} &= F_{By} \cdot (l_3 - z_3) \quad M_{x3}(0) = 186\text{Nm} \quad M_{x3}(l_3) = 0 \\
 M_{y3} &= -F_{Bx} \cdot (l_3 - z_3) \quad M_{y3}(0) = -533\text{Nm} \quad M_{y3}(l_3) = 0 \\
 M_{z3} &= 0
 \end{aligned}$$

y-z-Ebene

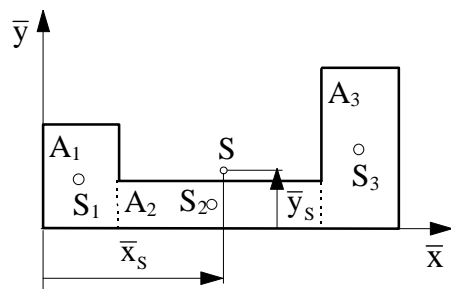
x-z-Ebene



$M_{b\max}$  bei  $z_2 = l_2 = 70\text{mm}$

$$M_{b\max} = \sqrt{M_{x2}^2(z_2 = l_2) + M_{y2}^2(z_2 = l_2)} = \sqrt{383^2 + 533^2} \text{Nm} = 656\text{Nm}$$

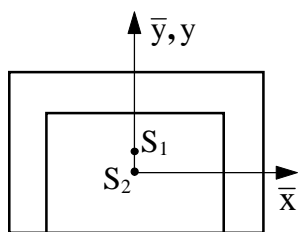
### Lösung 5.4



i	$A_i [\text{mm}^2]$	$\bar{x}_{Si} [\text{mm}]$	$\bar{y}_{Si} [\text{mm}]$	$\bar{x}_{Si} A_i [\text{mm}^3]$	$\bar{y}_{Si} A_i [\text{mm}^3]$
1	600	10	15	6000	9000
2	900	50	7,5	45000	6750
3	1100	90	27,5	99000	30250
$\Sigma$	2600			150000	46000

Damit  $\bar{x}_{SF} = \frac{150000}{2600} \text{ mm} = 57,5 \text{ mm};$   $\bar{y}_{SF} = \frac{46000}{2600} \text{ mm} = 17,7 \text{ mm};$

### Lösung 6.2



i	$A_i [\text{h}^2]$	$\bar{y}_{Si} [\text{h}]$	$\bar{y}_{Si} A_i [\text{h}^3]$	$I_{xxi} [\text{h}^4]$	$I_{yyi} [\text{h}^4]$	$\bar{y}_{Si}^2 A_i [\text{h}^4]$
1	40	0,5	20	83,33	213,33	10
2	-24	0	0	-32	-72	0
$\Sigma$	16		20	51,33	141,33	10

$\bar{y}_S = \frac{20}{16} \text{ h} = 1,25 \text{ h};$   $I_{xx} = 61,33 \text{ h}^4;$   $I_{xx} = \left( 61,33 - \frac{25}{16} \cdot 16 \right) \text{ h}^4 = 36,33 \text{ h}^4;$   $I_{yy} = 141,33 \text{ h}^4$

$I_{xy} = 0;$   $I_1 = I_{yy};$   $I_2 = I_{xx}$

### Lösung 6.6

i	$A_i [a^2]$	$\bar{x}_{Si} [a]$	$\bar{y}_{Si} [a]$	$\bar{x}_{Si} A_i [a^3]$	$\bar{y}_{Si} A_i [a^3]$
1	15	-1,5	2,5	-22,5	37,5
2	7,5	1	1,667	7,5	12,5
$\Sigma$	22,5			-15	50

$$\bar{x}_s = -\frac{2}{3}a = -0,667a; \quad \bar{y}_s = \frac{20}{9}a = 2,222a;$$

Fortsetzung der Tabelle:

$I_{xxi} [a^4]$	$I_{yyi} [a^4]$	$I_{xyi} [a^4]$	$\bar{y}_{Si}^2 A_i [a^4]$	$\bar{x}_{Si}^2 A_i [a^4]$	$\bar{x}_{Si} \bar{y}_{Si} A_i [a^4]$
31,25	11,25	0	93,75	33,75	-56,25
10,42	3,75	3,125	20,84	7,5	12,5
41,67	15	3,125	114,59	41,25	-43,75

$$I_{\bar{x}\bar{x}} = (41,67 + 114,59)a^4 = 156,26a^4; \quad I_{\bar{y}\bar{y}} = (15 + 41,25)a^4 = 56,25a^4$$

$$I_{\bar{x}\bar{y}} = (3,125 + 43,75)a^4 = 46,875a^4$$

$$I_{xx} = I_{\bar{x}\bar{x}} - \bar{y}_s^2 A = (156,26 - 111,09)a^4 = 45,17a^4$$

$$I_{yy} = I_{\bar{y}\bar{y}} - \bar{x}_s^2 A = (56,25 - 10)a^4 = 46,25a^4$$

$$I_{xy} = I_{\bar{x}\bar{y}} + \bar{x}_s \bar{y}_s A = (46,875 - 33,317)a^4 = 13,56a^4$$

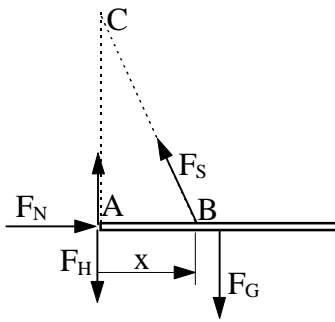
Hauptträgheitsmomente und Hauptachsen:

$$I_{1,2} = (45,71 \pm \sqrt{(-0,54)^2 + (13,56)^2})a^4 = (45,71 \pm 13,57)a^4$$

$$I_1 = 59,28a^4; \quad I_2 = 32,14a^4$$

$$\tan \varphi_{01} = \frac{I_1 - I_{xx}}{I_{xy}} = \frac{59,28 - 45,17}{13,56} = 1,04056 \quad \varphi_{01} = 46,1^\circ$$

### Lösung 7.3



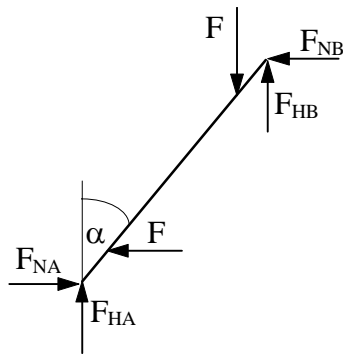
$$\curvearrowright \text{C: } F_G \cdot \frac{a}{2} - F_N \cdot a = 0 \quad F_N = \frac{1}{2} F_G$$

$$\curvearrowright \text{B: } \pm F_H \cdot x - F_G \left( \frac{a}{2} - x \right) = 0$$

$$F_{H\max} = \mu_0 F_N = \frac{1}{2} \mu_0 F_G$$

$$\pm \frac{1}{2} \mu_0 F_G \cdot x - F_G \left( \frac{a}{2} - x \right) = 0 \quad \pm \frac{1}{2} \mu_0 \cdot x - \frac{a}{2} + x = 0$$

### Lösung 7.5



$$\rightarrow: F_{NA} - F - F_{NB} = 0$$

$$\uparrow: F_{HA} - F + F_{HB} = 0$$

$$\curvearrowright \text{A: } F \cdot \frac{l}{8} \cos \alpha - F \cdot \frac{7}{8} l \sin \alpha + F_{NB} \cdot l \cos \alpha + F_{HB} \cdot l \sin \alpha = 0$$

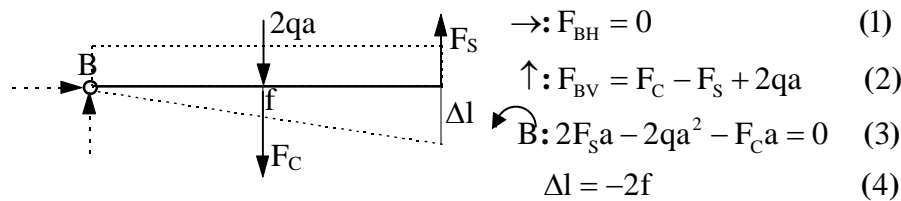
$$F_{HA\max} = \mu_0 F_{NA} \quad F_{HB\max} = \mu_0 F_{NB}$$

$$\mu_0^2 + \frac{3}{4} (1 + \cot \alpha) \mu_0 - \cot \alpha = 0$$

$$\alpha = 45^\circ: \quad \mu_0^2 + \frac{3}{2} \mu_0 - 1 = 0 \quad \mu_{01} = 0,5; (\mu_{02} = -2)$$

# Lösungen für Aufgaben zur Technischen Mechanik – Festigkeitslehre -

## Lösung 2.16



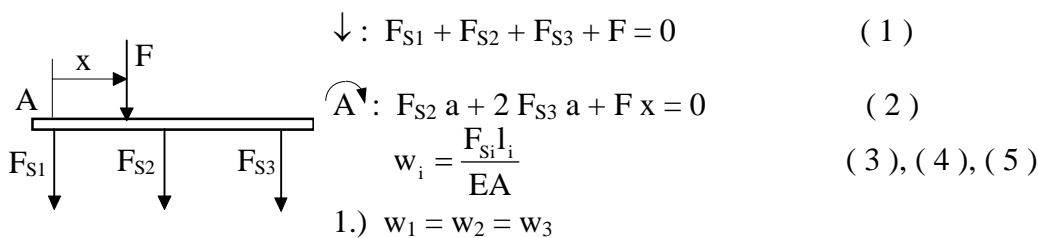
Für die lineare Feder gilt:  $f = \frac{F_C}{c}$  und damit wird aus Gleichungen (3) und (4)

$$2F_S - F_C = 2qa$$

$$\frac{F_S l}{EA} + 2 \frac{F_C}{c} = 0$$

$$F_S = \frac{qa}{1 + \frac{cl}{4EA}}; \quad F_C = -\frac{2qa \frac{cl}{4EA}}{1 + \frac{cl}{4EA}}; \quad F_{BV} = \frac{qa}{1 + \frac{cl}{4EA}}$$

## Lösung 2.22



$$\frac{F_{S1} a}{EA} = \frac{3F_{S2} a}{2EA} = \frac{2F_{S3} a}{EA} \quad F_{S2} = \frac{2}{3} F_{S1} \quad F_{S3} = \frac{1}{2} F_{S1} \quad (6), (7)$$

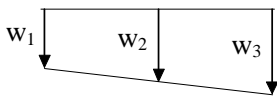
(6) und (7) in (1):  $F_{S1} + \frac{2}{3} F_{S1} + \frac{1}{2} F_{S1} = -F \quad F_{S1} = -\frac{6}{13} F$

(6) und (7) in (2):  $\frac{2}{3} F_{S1} a + F_{S1} a = -Fx \quad \boxed{x = -\frac{5 F_{S1}}{3 F} a = \frac{10}{13} a}$

2.)

$\sigma_1 = \frac{F_{S1}}{A} = -\frac{6}{13} \frac{F}{A}$	$\sigma_2 = \frac{F_{S2}}{A} = -\frac{4}{13} \frac{F}{A}$	$\sigma_3 = \frac{F_{S3}}{A} = -\frac{3}{13} \frac{F}{A}$
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3.)



$$w_2 = \frac{1}{2}(w_1 + w_3) \quad (6')$$

(1) bis (5) und (6') liefern  $F_{S1} + F_{S2} + F_{S3} = -F$  (1')

$$F_{S2}a + 2F_{S3}a = -\frac{2}{3}Fa \quad (2')$$

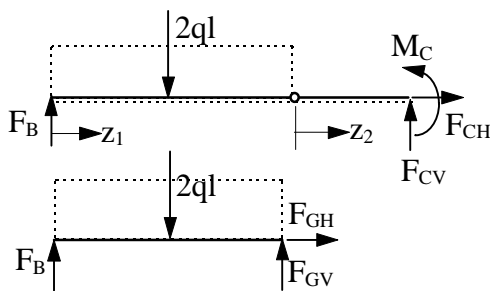
$$\frac{3}{2}F_{S2} = \frac{1}{2}F_{S1} + F_{S3} \quad (6')$$

$$F_{S1} + F_{S2} + F_{S3} = -F \quad (I) \quad F_{S1} - 3F_{S2} + 2F_{S3} = 0 \quad (II) \quad F_{S2} + 2F_{S3} = -\frac{2}{3}F \quad (III)$$

$$I - II: 4F_{S2} - F_{S3} = -F \quad III + 2 \cdot (I - II): 9F_{S2} = -\frac{8}{3}F$$

$F_{S2} = -\frac{8}{27}F$	$F_{S3} = -\frac{5}{27}F$	$F_{S1} = -\frac{14}{27}F$
$w_2 = -\frac{4}{9} \frac{Fa}{EA}$	$w_3 = -\frac{10}{27} \frac{Fa}{EA}$	$w_1 = -\frac{14}{27} \frac{Fa}{EA}$

#### Lösung 4.1



$$\rightarrow: F_{CH} = 0$$

$$\uparrow: F_B + F_{CV} - 2ql = 0$$

$$C: 3F_B l - 4ql^2 - M_C = 0$$

$$G: 2F_B l - 2ql^2 = 0$$

$$F_B = ql; \quad F_{CH} = 0; \quad F_{CV} = ql; \quad M_C = -ql^2$$

$$F_{GH} = 0; \quad F_{GV} = ql$$

$$0 \leq z_1 \leq 2l$$

$$0 \leq z_2 \leq l$$

$$F_L(z_1) = 0$$

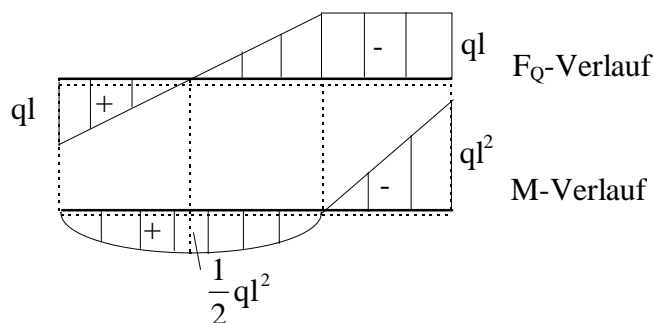
$$F_L(z_2) = 0$$

$$F_Q(z_1) = q(1 - z_1)$$

$$F_Q(z_2) = -ql$$

$$M(z_1) = qlz_1 - \frac{1}{2}qz_1^2$$

$$M(z_2) = -qlz_2$$



Aufgabe 6.2 (Statik):  $I_{xx} = 36,33h^4$

Maximalwerte für y: Unterseite  $y = e_1 = 3,25h$

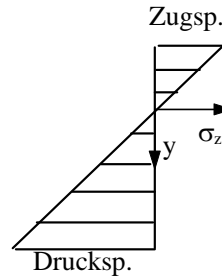
Oberseite  $y = -e_2 = -1,75h$

Spannungsverteilung an der Einspannstelle (Maximales Moment)

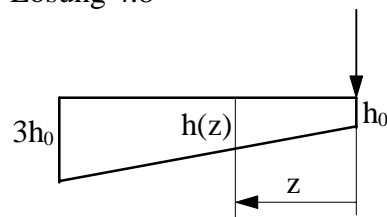
$$\sigma_z(y) = -\frac{ql^2}{I_{xx}} y$$

$$\sigma_z(y = e_1) = -0,0894 \frac{ql^2}{h^3} \Rightarrow \text{Druckspannung}$$

$$\sigma_z(y = -e_2) = 0,0482 \frac{ql^2}{h^3} \Rightarrow \text{Zugspannung}$$



#### Lösung 4.8



$$|\sigma(z)|_{\max} = \frac{|M(z)|}{W_x(z)} \quad |M(z)| = F \cdot z \quad W_x(z) = \frac{bh^2(z)}{6}$$

$$h(z) = h_0 \left(1 + 2 \frac{z}{l}\right) \quad W_x(z) = \frac{bh_0^2}{6} \left(1 + 2 \frac{z}{l}\right)^2$$

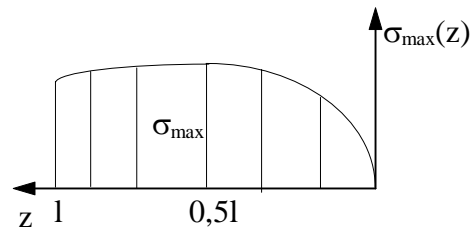
$$|\sigma(z)|_{\max} = \frac{6F \cdot z}{bh_0^2 \left(1 + 2 \frac{z}{l}\right)^2}$$

Ort der max. Biegespannung entweder an der Einspannstelle (max. Moment) oder an der Stelle, an der  $\frac{d\sigma_{\max}(z)}{dz} = 0$  ist.

$$\frac{d\sigma_{\max}(z)}{dz} = \frac{6F}{bh_0^2} \frac{\left(1 + 2 \frac{z}{l}\right)^2 - 4 \frac{z}{l} \left(1 + 2 \frac{z}{l}\right)}{\left(1 + 2 \frac{z}{l}\right)^4} = 0$$

$$\left(1 + 2 \frac{z}{l}\right) - 4 \frac{z}{l} = 0 \quad \frac{z}{l} = \frac{1}{2}$$

$$\left|\sigma\left(\frac{l}{2}\right)\right|_{\max} = \frac{3}{4} \frac{Fl}{bh_0^2} \quad |\sigma(l)|_{\max} = \frac{2}{3} \frac{Fl}{bh_0^2}$$



Die absolut größte Spannung beträgt  $\sigma_{\max} = \frac{3}{4} \frac{Fl}{bh_0^2}$  und tritt an der Stelle  $z = \frac{l}{2}$  auf.

#### Lösung 4.18

Größte Beanspruchung an der Einspannstelle! Max. Spannung an einem Eckpunkt des Querschnittes, d.h.:



$$|\sigma_{\max}| = \frac{|M_{x\max}|}{I_{xx}} |y_{\max}| + \frac{|M_{y\max}|}{I_{yy}} |x_{\max}| = \frac{|M_{x\max}|}{W_x} + \frac{|M_{y\max}|}{W_y}$$

$$|M_{x\max}| = 2F \cdot 2l \quad W_x = \frac{bh^2}{6}$$

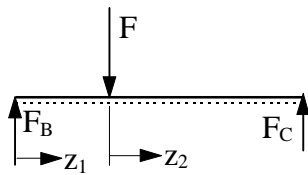
$$|M_{y\max}| = F \cdot l \quad W_y = \frac{hb^2}{6}$$

$$|\sigma_{\max}| = \frac{24Fl}{bh^2} + \frac{6Fl}{hb^2} = \sigma_{zul} \Rightarrow b^2 - \frac{24Fl}{\sigma_{zul}h^2} \cdot b - \frac{6Fl}{\sigma_{zul}h} = 0$$

$$b = \frac{12Fl}{\sigma_{zul}h^2} \left( 1 \pm \sqrt{1 + \frac{\sigma_{zul} \cdot h^3}{24Fl}} \right) \quad b = 58,7mm \quad b_{gew} = 60mm$$

Nur das positive Vorzeichen ist physikalisch sinnvoll.

### Lösung 4.33



$$F_B = \frac{2}{3}F; \quad F_C = \frac{1}{3}F$$

$$0 \leq z_1 \leq a \quad 0 \leq z_2 \leq 2a$$

$$M(z_1) = F_B z_1 = \frac{2}{3}Fz_1 \quad M(z_2) = F_C(2a - z_2) = \frac{1}{3}F(2a - z_2)$$

$$EIv''(z_1) = -\frac{2}{3}Fz_1$$

$$EIv''(z_2) = -\frac{1}{3}F(2a - z_2)$$

$$EIv'(z_1) = -\frac{1}{3}Fz_1^2 + C_1$$

$$EIv'(z_2) = \frac{1}{6}F(2a - z_2)^2 + C_3$$

$$EIv(z_1) = -\frac{1}{9}Fz_1^3 + C_1z_1 + C_2$$

$$EIv(z_2) = -\frac{1}{18}F(2a - z_2)^3 + C_3z_2 + C_4$$

$$\text{RB/ÜB } v(z_1 = 0) = 0 \Rightarrow C_2 = 0$$

$$v(z_2 = 2a) = 0 \Rightarrow 2C_3a + C_4 = 0$$

$$v(z_1 = a) = v(z_2 = 0) \Rightarrow -\frac{1}{9}Fa^3 + C_1a = -\frac{4}{9}Fa^3 + C_4$$

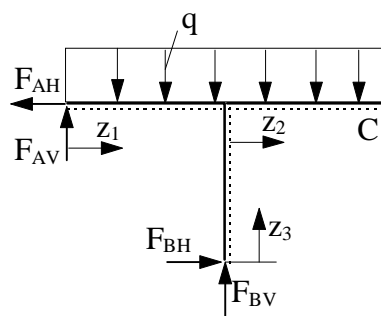
$$v'(z_1 = a) = v'(z_2 = 0) \Rightarrow -\frac{1}{3}Fa^2 + C_1 = \frac{2}{3}Fa^2 + C_3$$

$$C_1 = \frac{5}{9}Fa^2 \quad C_2 = 0 \quad C_3 = -\frac{4}{9}Fa^2 \quad C_4 = \frac{8}{9}Fa^3$$

$$v(z_1) = \frac{Fa^3}{9EI} \left[ 5 \frac{z_1}{a} - \left( \frac{z_1}{a} \right)^3 \right] \quad v(z_2) = \frac{Fa^3}{18EI} \left[ 8 + 4 \frac{z_2}{a} - 6 \left( \frac{z_2}{a} \right)^2 + \left( \frac{z_2}{a} \right)^3 \right]$$

$$v_F = v(z_1 = a) = v(z_2 = 0) = \frac{4Fa^3}{9EI}$$

### Lösung 4.45



Die Aufgabe ist einfach statisch unbestimmt.

Gleichgewichtsbedingungen:

$$\leftarrow : F_{AH} - F_{BH} = 0$$

$$\uparrow : F_{AV} + F_{BV} - 2qa = 0$$

$$\curvearrowright (B) : F_{AV} \cdot a - F_{AH} \cdot a = 0$$

$$M(z_1) = F_{AV} \cdot z_1 - \frac{1}{2} qz_1^2$$

$$M(z_2) = -\frac{1}{2} q(a - z_2)^2$$

$$EIv''(z_1) = \frac{1}{2} qz_1^2 - F_{AV} \cdot z_1$$

$$EIv''(z_2) = \frac{1}{2} q(a - z_2)^2$$

$$EIv'(z_1) = \frac{1}{6} qz_1^3 - \frac{1}{2} F_{AV} \cdot z_1^2 + C_1$$

$$EIv'(z_2) = -\frac{1}{6} q(a - z_2)^3 + C_3$$

$$EIv(z_1) = \frac{1}{24} qz_1^4 - \frac{1}{6} F_{AV} \cdot z_1^3 + C_1 z_1 + C_2 \quad EIv(z_2) = \frac{1}{24} q(a - z_2)^4 + C_3 z_2 + C_4$$

$$M(z_3) = -F_{BH} \cdot z_3$$

$$EIv''(z_3) = F_{BH} \cdot z_3$$

$$EIv'(z_3) = \frac{1}{2} F_{BH} \cdot z_3^2 + C_5$$

$$EIv(z_3) = \frac{1}{6} F_{BH} \cdot z_3^3 + C_5 z_3 + C_6$$

RB/ÜB:

$$1. \quad v(z_1 = 0) = 0 \Rightarrow C_2 = 0$$

$$2. \quad v(z_3 = 0) = 0 \Rightarrow C_6 = 0$$

$$3. \quad v(z_1 = a) = 0 \Rightarrow \frac{1}{24} qa^4 - \frac{1}{6} F_{AV} a^3 + C_1 a = 0 \quad C_1 = \frac{1}{6} F_{AV} a^2 - \frac{1}{24} qa^3$$

$$4. \quad v(z_2 = 0) = 0 \Rightarrow \frac{1}{24} qa^4 + C_4 = 0 \quad C_4 = -\frac{1}{24} qa^4$$

$$5. \quad v(z_3 = a) = 0 \Rightarrow \frac{1}{6} F_{BH} a^3 + C_5 a = 0 \quad C_5 = -\frac{1}{6} F_{BH} a^2$$

$$6. \quad v'(z_1 = a) = v'(z_2 = 0) \Rightarrow \frac{1}{6} qa^3 - \frac{1}{2} F_{AV} a^2 + C_1 = -\frac{1}{6} qa^3 + C_3$$

$$7. \quad v'(z_3 = a) = v'(z_2 = 0) \Rightarrow \frac{1}{2} F_{BH} a^2 + C_5 = -\frac{1}{6} qa^3 + C_3 \quad C_3 = \frac{1}{3} F_{BH} a^2 + \frac{1}{6} qa^3$$

Ergebnisse aus GGW und RB/ÜB:

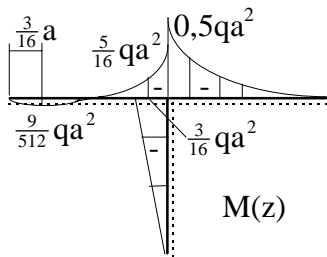
$$\text{aus 6. folgt mit den GGW } F_{AV} = \frac{3}{16} qa \Rightarrow F_{AH} = F_{BH} = \frac{3}{16} qa \text{ und } F_{BV} = \frac{29}{16} qa$$

Momentenverlauf:

$$M(z_1) = \frac{1}{2} q z_1 \left( \frac{3}{8} a - z_1 \right) \quad F_Q(z_1^*) = \frac{3}{16} qa - q z_1^* = 0 \quad z_1^* = \frac{3}{16} a \quad M(z_1^*) = \frac{9}{512} qa^2$$

$$M(z_2) = -\frac{1}{2} q (a - z_2)^2$$

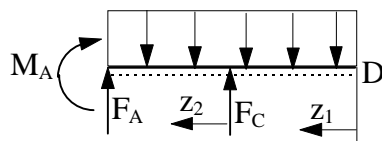
$$M(z_3) = -\frac{3}{16} q a z_3$$



Verschiebung des Punktes C:

$$v_C = v(z_2 = a) = \frac{1}{EI} (C_3 a + C_4) = \frac{3qa^4}{16EI}$$

#### Lösung 4.50



$$F_C = v(z_2 = 0) \cdot c$$

$$M(z_1) = -\frac{1}{2} q z_1^2$$

$$M(z_2) = -\frac{1}{2} q (a + z_2)^2 + F_C z_2$$

$$EI v''(z_1) = \frac{1}{2} q z_1^2$$

$$EI v''(z_2) = \frac{1}{2} q (a + z_2)^2 - F_C z_2$$

$$EI v'(z_1) = \frac{1}{6} q z_1^3 + C_1$$

$$EI v'(z_2) = \frac{1}{6} q (a + z_2)^3 - \frac{1}{2} F_C z_2^2 + C_3$$

$$EI v(z_1) = \frac{1}{24} q z_1^4 + C_1 z_1 + C_2$$

$$EI v(z_2) = \frac{1}{24} q (a + z_2)^4 - \frac{1}{6} F_C z_2^3 + C_3 z_2 + C_4$$

RB/ÜB:

$$1. \quad v'(z_2 = a) = 0 \quad \Rightarrow \quad C_3 = \frac{1}{2} F_C a^2 - \frac{4}{3} qa^3$$

$$2. \quad v(z_2 = a) = 0 \quad \Rightarrow \quad C_4 = \frac{2}{3} qa^4 - \frac{1}{3} F_C a^3$$

$$3. \quad v'(z_1 = a) = v'(z_2 = 0) \quad \Rightarrow \quad C_1 = C_3 = \frac{1}{2} F_C a^2 - \frac{4}{3} qa^3$$

$$4. \quad v(z_1 = a) = v(z_2 = 0) \quad \Rightarrow \quad C_2 = C_4 - C_1 a = 2qa^4 - \frac{5}{6} F_C a^3$$

$$F_C = v(z_2 = 0) \cdot c = \left( \frac{17qa^4}{24EI} - \frac{F_C a^3}{3EI} \right) \cdot c \Rightarrow F_C = \frac{17qac}{8 \left( \frac{3EI}{a^3} + c \right)}$$

$$F_C(c \rightarrow \infty) = \frac{17}{8} qa$$

$$v_D = v(z_1 = 0) = \frac{C_2}{EI} = \frac{2qa^4}{EI} - \frac{5F_C a^3}{6EI}$$

$$v_D(c \rightarrow 0) = \frac{2qa^4}{EI} \quad v_D(c \rightarrow \infty) = \frac{2qa^4}{EI} - \frac{85qa^4}{48EI} = \frac{11qa^4}{48EI}$$

### Lösung 5.6

$$1. \quad m_v = A_v \cdot \rho \cdot l = \frac{\pi D^2}{4} \cdot \rho \cdot l \quad m_h = A_h \cdot \rho \cdot l = \frac{\pi}{4} (D^2 - d^2) \cdot \rho \cdot l$$

$$\frac{m_v - m_h}{m_v} \cdot 100\% = \left( \frac{d}{D} \right)^2 \cdot 100\% = 25\%$$

$$2. \quad \tau_v = \frac{M_t}{W_{tv}} \quad \tau_h = \frac{M_t}{W_{th}} \quad W_{tv} = \frac{\pi D^3}{16} \quad W_{th} = W_{tv} \left( 1 - \left( \frac{d}{D} \right)^4 \right)$$

$$\left( \frac{\tau_h - \tau_v}{\tau_v} \right) \cdot 100\% = \left( \frac{W_{tv}}{W_{th}} - 1 \right) \cdot 100\% = \left( \frac{1}{1 - \left( \frac{d}{D} \right)^4} - 1 \right) \cdot 100\% = \left( \frac{16}{15} - 1 \right) \cdot 100\% = 6,7\%$$

$$3. \quad \left( \frac{g_h - g_v}{g_v} \right) \cdot 100\% = \left( \frac{I_{tv}}{I_{th}} - 1 \right) \cdot 100\% = \left( \frac{1}{1 - \left( \frac{d}{D} \right)^4} - 1 \right) \cdot 100\% = \left( \frac{16}{15} - 1 \right) \cdot 100\% = 6,7\%$$

### Lösung 7.11

Momente an der Einspannstelle: Biege- und Torsionsmoment erreichen an der Einspannstelle ihr Maximum.

$$|M_{bmax}| = |(2F_1 - F_2) \cdot a| = 80 \cdot 10^4 \text{ Nmm}$$

$$|M_{tmax}| = |(F_1 - F_2) \cdot b| = 60 \cdot 10^4 \text{ Nmm}$$

$$\text{Spannungen: } \sigma_{max} = \frac{|M_{bmax}| \cdot 32 \cdot D_a}{\pi(D_a^4 - D_i^4)} \quad \tau_{max} = \frac{|M_{tmax}| \cdot 16 \cdot D_a}{\pi(D_a^4 - D_i^4)}$$

Der erforderliche Innendurchmesser folgt aus der Beziehung  $\sigma_{v4} = \sigma_{max} \sqrt{1 + 3 \left( \frac{\tau_{max}}{\sigma_{max}} \right)^2} \leq \sigma_{zul}$

$$\sigma_{zul} \geq \frac{32D_a |M_{bmax}|}{\pi(D_a^4 - D_i^4)} \sqrt{1 + 3 \left( \frac{|M_{tmax}|}{2|M_{bmax}|} \right)^2}$$

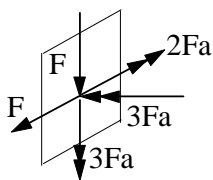
$$D_i \leq \sqrt[4]{D_a^4 - \frac{32D_a |M_{bmax}|}{\pi \sigma_{zul}} \sqrt{1 + 3 \left( \frac{|M_{tmax}|}{2|M_{bmax}|} \right)^2}} = 42,3 \text{ mm}$$

Gewählt:  $D_i = 42 \text{ mm}$

Spannungsnachweis:  $\sigma_{vorh} = 154,8 \frac{\text{N}}{\text{mm}^2} < \sigma_{zul} = 160 \frac{\text{N}}{\text{mm}^2}$

### Lösung 7.13

Biege- und Torsionsmomente an der Einspannstelle:



$$F_N = 0$$

$$|M_x| = 2Fa \quad |M_y| = 3Fa \quad |M_t| = 3Fa$$

$$M_{res} = \sqrt{M_x^2 + M_y^2} = Fa \sqrt{4 + 9} = \sqrt{13} Fa = 1,8028 \cdot 10^6 \text{ Nmm}$$

$$\sigma_{max} = \frac{32M_{res} \cdot d_a}{\pi(d_a^4 - d_i^4)} \quad \tau_{max} = \frac{16M_t \cdot d_a}{\pi(d_a^4 - d_i^4)}$$

Spannungen:

$$\sigma_{vmax} = \sqrt{\sigma_{max}^2 + 3\tau_{max}^2} = \frac{32}{\pi} \frac{d_a}{(d_a^4 - d_i^4)} \sqrt{M_{res}^2 + 3 \left( \frac{M_t}{2} \right)^2}$$

Erforderlicher Innendurchmesser:

Aus  $\sigma_{vmax} \leq \sigma_{zul}$  folgt

$$d_i \leq \sqrt[4]{d_a^4 - \frac{32}{\pi} \frac{d_a}{\sigma_{zul}} \sqrt{M_{res}^2 + 3 \left( \frac{M_t}{2} \right)^2}} = 94,9 \text{ mm} \quad d_{igew} = 94 \text{ mm}$$

$$\sigma_{vorh} = 103,24 \frac{\text{N}}{\text{mm}^2} < \sigma_{zul} = 120 \frac{\text{N}}{\text{mm}^2}$$

## Lösungen für Aufgaben zur Technischen Mechanik – Dynamik -

### Lösung 1.6

Bereich I:  $\ddot{x} = a_A \quad \dot{x} = a_A t + C_1 \quad x = \frac{1}{2} a_A t^2 + C_1 t + C_2$

AB:  $t = 0 \quad \dot{x} = 0, \quad x = 0 \Rightarrow C_1 = 0, \quad C_2 = 0$

$t = t_1: \quad \dot{x}(t_1) = v, \quad x(t_1) = s_1 \Rightarrow t_1 = \frac{v}{a_A} = 20\text{s} \quad s_1 = \frac{1}{2} a_A t_1^2 = 160\text{m}$

Bereich II:  $\ddot{x} = 0 \quad \dot{x} = C_3 \quad x = C_3 t + C_4$

ÜB  $t = t_1 \quad \dot{x} = v, \quad x = s_1 \Rightarrow C_3 = v = 16 \frac{\text{m}}{\text{s}}, \quad C_4 = s_1 - v t_1 = -160\text{m}$

$t = t_2: \quad \dot{x}(t_2) = v, \quad x(t_2) = s_2 = v t_2 + s_1 - v t_1 = v(t_2 - t_1) + s_1$

Bereich III:  $\ddot{x} = a_B \quad \dot{x} = a_B t + C_5 \quad x = \frac{1}{2} a_B t^2 + C_5 t + C_6$

ÜB  $t = t_2 \quad \dot{x} = v, \quad x = s_2 \Rightarrow C_5 = v - a_B t_2 \quad C_6 = s_2 - v t_2 + \frac{1}{2} a_B t_2^2$

Endbedingung:  $t = t_3 \quad \dot{x} = 0, \quad x = s_3 = S \Rightarrow a_B(t_3 - t_2) + v = 0 \quad (1) \text{ und}$

$S = \frac{1}{2} a_B t_3^2 + (v - a_B t_2) t_3 + s_2 - v t_2 + \frac{1}{2} a_B t_2^2 = \frac{1}{2} a_B (t_3 - t_2)^2 + v(t_3 - t_2) + s_2 \quad (2)$

$t_2 = t_3 + \frac{v}{a_B} \text{ in } (2) \Rightarrow t_3 = t_1 + \frac{S - s_1}{v} - \frac{1}{2} \frac{v}{a_B} = (20 + 23 + 8)\text{s} = 51\text{s} = t_{\text{ges}}$

$t_2 = (51 - 16)\text{s} = 35\text{s} \Rightarrow s_2 = S - \frac{1}{2} a_B (t_3 - t_2)^2 - v(t_3 - t_2) = 400\text{m}$

### Lösung 1.7

1. Bereich:  $0 \leq t \leq t_1$

$a(t) = K_1 t^2 \quad v(t) = \frac{1}{3} K_1 t^3 + C_1 \quad s(t) = \frac{1}{12} K_1 t^4 + C_1 t + C_2$

$t = 0: \quad v(0) = 0 \quad s(0) = 0 \Rightarrow C_1 = 0 \quad C_2 = 0$

$t = t_1: \quad a(t_1) = a_1 \Rightarrow a_1 = K_1 t_1^2 \quad K_1 = \frac{a_1}{t_1^2} = 1 \frac{\text{m}}{\text{s}^4}$

$v(t_1) = \frac{1}{3} K_1 t_1^3 = \frac{1}{3} \frac{\text{m}}{\text{s}} \quad s(t_1) = \frac{1}{12} K_1 t_1^4 = \frac{1}{12} \text{m}$

2. Bereich  $t_1 \leq t \leq t_2$

$$a(t) = a_1 \quad v(t) = a_1 t + C_3 \quad s(t) = \frac{1}{2} a_1 t^2 + C_3 t + C_4$$

$$\text{ÜB } t = t_1 \quad v_2(t_1) = v_1(t_1) = \frac{1}{3} \frac{m}{s} \Rightarrow \frac{1}{3} \frac{m}{s} = a_1 t_1 + C_3 = 1 \frac{m}{s} + C_3 \quad C_3 = -\frac{2}{3} \frac{m}{s}$$

$$s_2(t_1) = s_1(t_1) = \frac{1}{12} m \Rightarrow \frac{1}{12} m = \frac{1}{2} m - \frac{2}{3} m + C_4 \quad C_4 = \frac{1}{4} m$$

$$t = t_2 \quad v_2(t_2) = a_1 t_2 + C_3 = 2 \frac{m}{s} - \frac{2}{3} \frac{m}{s} = \frac{4}{3} \frac{m}{s}$$

$$s_2(t_2) = \frac{1}{2} a_1 t_2^2 + C_3 t_2 + C_4 = 2m - \frac{4}{3} m + \frac{1}{4} m = \frac{11}{12} m$$

3. Bereich  $t_2 \leq t \leq t_3$

$$a(t) = K_2 + K_3 t \quad v(t) = K_2 t + \frac{1}{2} K_3 t^2 + C_5 \quad s(t) = \frac{1}{2} K_2 t^2 + \frac{1}{6} K_3 t^3 + C_5 t + C_6$$

$$t = t_2: \quad a(t_2) = a_1 \quad a_1 = K_2 + K_3 t_2 \quad | -$$

$$t = t_3: \quad a(t_3) = 0 \quad 0 = K_2 + K_3 t_3 \quad | +$$

$$-a_1 = K_3(t_3 - t_2) \Rightarrow K_3 = \frac{-a_1}{t_3 - t_2} = -1 \frac{m}{s^3} K_2 = -K_3 t_3 = 3 \frac{m}{s^2}$$

$$\text{ÜB: } t = t_2 \quad v_2(t_2) = v_3(t_2) = \frac{4}{3} \frac{m}{s} \Rightarrow \frac{4}{3} \frac{m}{s} = K_2 t_2 + \frac{1}{2} K_3 t_2^2 + C_5 = 6 \frac{m}{s} - 2 \frac{m}{s} + C_5$$

$$s_2(t_2) = s_3(t_2) = \frac{11}{12} m \Rightarrow \frac{11}{12} m = -\frac{4}{3} m + 6m + C_5 t_2 + C_6$$

$$C_5 = -\frac{8}{3} \frac{m}{s} \quad C_6 = \frac{19}{12} m$$

### Lösung 2.1

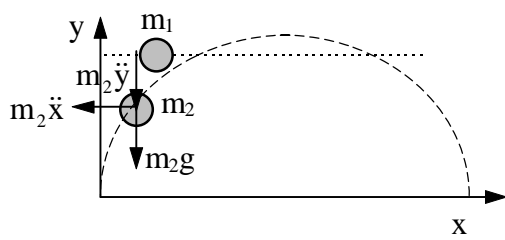
$$\omega_w \cdot \frac{D}{2} = v \Rightarrow \omega_w = \frac{2v}{D}$$

$$\omega_w \cdot r_G = \omega_{ZW} \cdot r_K \Rightarrow \omega_{ZW} = \frac{r_G}{r_K} \cdot \omega_w = \frac{r_G}{r_K} \cdot \frac{2v}{D}$$

$$\omega_{ZW} \cdot r_G = \omega_M \cdot r_K \Rightarrow \frac{r_G}{r_K} \cdot \frac{2v}{D} = \frac{r_K}{r_G} \cdot 2\pi n_M \quad \left( \frac{r_K}{r_G} \right)^2 = \frac{v}{\pi n_M D} = i^2$$

$$i = \frac{r_K}{r_G} = \sqrt{\frac{v}{\pi n_M D}} = 0,18 = \frac{z_K}{z_G}$$

### Lösung 3.5



$$y_1 = H_0 \quad \dot{x}_1 = v_1 \quad x_1(t) = v_1 t + C_1 \quad C_1 = 0$$

$$\ddot{y}_2 = -g \quad \dot{y}_2 = -gt + C_2 \quad y_2(t) = -\frac{1}{2}gt^2 + C_2 t + C_3$$

$$\ddot{x}_2 = 0 \quad \dot{x}_2 = C_4 \quad x_2(t) = C_4 t + C_5$$

$$AB: t = 0 \quad \dot{x}_2 = v_2 \cos \alpha \Rightarrow C_4 = v_2 \cos \alpha$$

$$\dot{y}_2 = v_2 \sin \alpha \Rightarrow C_2 = v_2 \sin \alpha$$

$$x_2 = 0 \Rightarrow C_5 = 0$$

$$y_2 = 0 \Rightarrow C_3 = 0$$

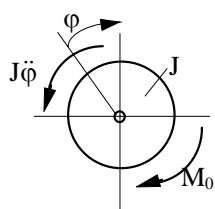
$$EB: t = t_E \quad x_1(t_E) = x_2(t_E) \Rightarrow v_2 = \frac{v_1}{\cos \alpha}$$

$$y_1(t_E) = y_2(t_E) \Rightarrow H_0 = -\frac{1}{2}gt_E^2 + v_1 \tan \alpha \cdot t_E$$

$$t_E^2 - 2 \frac{v_1 \tan \alpha}{g} t_E + 2 \frac{H_0}{g} = 0 \Rightarrow t_{E1,2} = \frac{v_1 \tan \alpha}{g} \pm \sqrt{\left(\frac{v_1 \tan \alpha}{g}\right)^2 - 2 \frac{H_0}{g}}$$

$$\text{Zahlenwerte: } v_2 = 1200 \frac{\text{km}}{\text{h}}, \quad t_{E1} = 3,67\text{s} \quad (t_{E2} = 55,1\text{s})$$

### Lösung 3.13



$$J\ddot{\varphi} = M_0 \quad J(\omega - \omega_0) = M_0(t - t_0)$$

$$t_0 = 0 \quad \omega_0 = 0$$

$$t = t_1: J\omega_1 = M_0 t_1 \Rightarrow M_0 = \frac{J\omega_1}{t_1}$$

$$J = \frac{1}{2}m_1\left(\frac{D}{2}\right)^2 - 6\left\{\frac{1}{2}m_2\left(\frac{d}{2}\right)^2 + m_2e^2\right\}$$

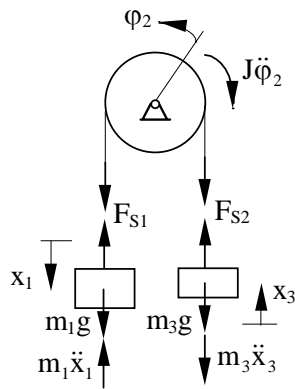
$$m_1 = \rho a \frac{\pi D^2}{4} \quad m_2 = \rho b \frac{\pi d^2}{4} \Rightarrow J = \frac{\rho \pi}{32} [aD^4 - 6bd^2(d^2 + 8e^2)]$$

$$M_0 = \frac{\omega_1 \rho \pi}{32 t_1} [aD^4 - 6bd^2(d^2 + 8e^2)] \quad n_1 = \frac{\omega_1}{2\pi}$$

$$\text{Zahlenwerte: } n_1 = 4,77\text{s}^{-1} \equiv 286,5\text{min}^{-1} \quad M_0 = 0,3933\text{Nm}$$



### Lösung 3.20



ZB:  $x_3 = x_1 = x \quad \varphi_2 = \varphi = \frac{x}{r}$

$$J = \frac{1}{2} m_2 r^2$$

$$F_{S1} = m_1 g - m_1 \ddot{x}_1 = m_1 g - m_1 \ddot{x}$$

$$F_{S2} = m_3 g + m_3 \ddot{x}_3 = m_3 g + m_3 \ddot{x}$$

$$J \ddot{\varphi}_2 + F_{S2} r - F_{S1} r = 0 = J \frac{\ddot{x}}{r} + m_3 g r + m_3 \ddot{x} r - m_1 g r + m_1 \ddot{x} r$$

$$\ddot{x} = g \frac{m_1 - m_3}{m_1 + \frac{1}{2} m_2 + m_3}$$

$$T = \frac{1}{2} m_1 \dot{x}_1^2 + \frac{1}{2} m_3 \dot{x}_3^2 + \frac{1}{2} J \dot{\varphi}_2^2 = \frac{1}{2} \dot{x}^2 \left( m_1 + \frac{1}{2} m_2 + m_3 \right)$$

$$U = -m_1 g x_1 + m_3 g x_3 = -g x (m_1 - m_3)$$

$$L = T - U = \frac{1}{2} \dot{x}^2 \left( m_1 + \frac{1}{2} m_2 + m_3 \right) + g x (m_1 - m_3)$$

$$\frac{d}{dt} \left( \frac{\partial L}{\partial \dot{x}} \right) - \frac{\partial L}{\partial x} = 0$$

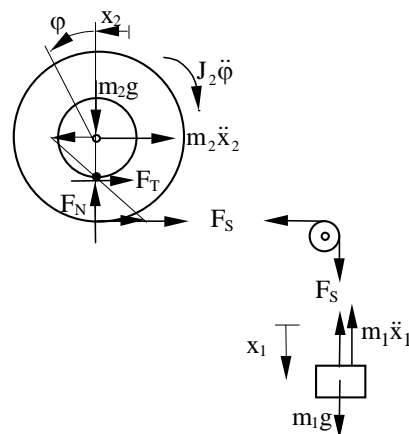
$$\frac{\partial L}{\partial x} = g(m_1 - m_3) \quad \frac{\partial L}{\partial \dot{x}} = \dot{x} \left( m_1 + \frac{1}{2} m_2 + m_3 \right) \quad \frac{d}{dt} \left( \frac{\partial L}{\partial \dot{x}} \right) = \ddot{x} \left( m_1 + \frac{1}{2} m_2 + m_3 \right)$$

$$T + U = \text{konst.} \quad \Rightarrow \quad \frac{d}{dt} (T + U) = 0$$

$$T + U = \frac{1}{2} \dot{x}^2 \left( m_1 + \frac{1}{2} m_2 + m_3 \right) - g x (m_1 - m_3)$$

$$\ddot{x} \left( m_1 + \frac{1}{2} m_2 + m_3 \right) - g \dot{x} (m_1 - m_3) = 0$$

### Lösung 3.26



$$\varphi = \frac{x_2}{r} \quad x_1 = (R - r)\varphi = \left( \frac{R}{r} - 1 \right) x_2$$

$$F_S = m_1 g - m_1 \left( \frac{R}{r} - 1 \right) \ddot{x}_2$$

$$F_S (R - r) - J_2 \frac{\ddot{x}_2}{r} - m_2 \ddot{x}_2 r = 0 \quad \left| \cdot \frac{1}{r} \right.$$

$$\ddot{x}_2 \left\{ m_2 + \frac{J_2}{r^2} + m_1 \left( \frac{R}{r} - 1 \right)^2 \right\} = m_1 g \left( \frac{R}{r} - 1 \right)$$

$$J_2 = 2J_A + J_B \quad J_A = \frac{1}{2} m_A r^2 \quad J_B = \frac{1}{2} m_B R^2$$

$$m_2 = 2m_A + m_B \quad m_A = \pi r^2 \frac{d}{2} \cdot \rho \quad m_B = \pi R^2 b \cdot \rho$$

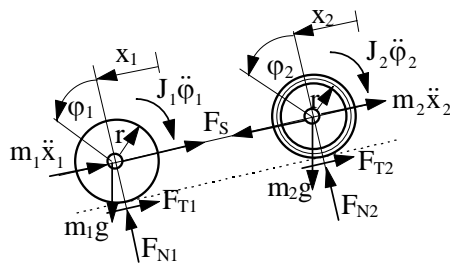
$$J_2 = \frac{1}{2} \pi \rho \{ dr^4 + bR^4 \} \quad m_2 = \pi \rho \{ dr^2 + bR^2 \} \Rightarrow \rho = \frac{m_2}{\pi \{ dr^2 + bR^2 \}}$$

$$J_2 = \frac{1}{2} m_2 \frac{\{ dr^4 + bR^4 \}}{\{ dr^2 + bR^2 \}}$$

$$\ddot{x}_1 = \left( \frac{R}{r} - 1 \right) \ddot{x}_2 = \frac{m_1 g}{m_1 + \frac{m_2}{\left( \frac{R}{r} - 1 \right)^2} + \frac{J_2}{r^2 \left( \frac{R}{r} - 1 \right)^2}}$$

$$T = \frac{1}{2} (m_2 \dot{x}_2^2 + m_1 \dot{x}_1^2 + J_2 \dot{\varphi}^2) \quad U = -m_1 g x_1 \quad \text{z.B.} \quad \frac{d}{dt} (T + U) = 0$$

### Lösung 3.28



$$x_1 = x_2 = x \quad \varphi_1 = \varphi_2 = \varphi = \frac{x}{r}$$

$$m_1: J_1 \ddot{\varphi} + F_S r + m_1 \ddot{x} r - m_1 g \sin \alpha \cdot r = 0$$

$$m_2: J_2 \ddot{\varphi} - F_S r + m_2 \ddot{x} r - m_2 g \sin \alpha \cdot r = 0$$

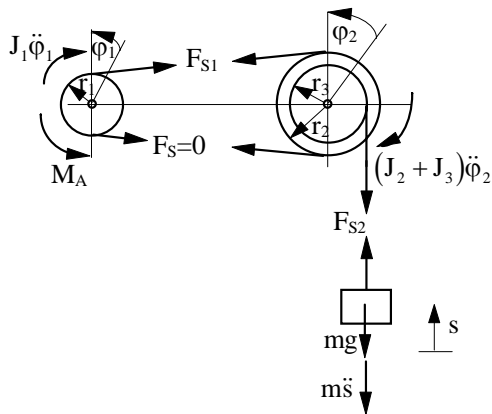
$$J_2 \frac{\ddot{x}}{r^2} + m_2 \ddot{x} - m_2 g \sin \alpha + J_1 \frac{\ddot{x}}{r^2} + m_1 \ddot{x} - m_1 g \sin \alpha = 0$$

$$m_1 = m_2 = m \quad J_1 = \frac{1}{2} m r^2 \quad J_2 = m r^2$$

$$\ddot{x} \left( m + m + \frac{1}{2} m + m \right) = 2 m g \sin \alpha \quad \ddot{x} = \frac{4}{7} g \sin \alpha$$

$$F_S = m(\ddot{x} + \ddot{x} - g \sin \alpha) = \frac{1}{7} m g \sin \alpha$$

### Lösung 3.31



$$\text{ZB: } \varphi_2 = \frac{s}{r_3} \quad r_2 \varphi_2 = r_1 \varphi_1 \Rightarrow \varphi_1 = \frac{r_2}{r_1} \frac{s}{r_3}$$

$$\text{Masse } m: m \ddot{s} + mg - F_{S2} = 0$$

$$\text{Scheibe 1: } J_1 \ddot{\varphi}_1 - M_A + F_{S1} r_1 = 0$$

$$\text{Scheibe 2/3: } (J_2 + J_3) \ddot{\varphi}_2 - F_{S1} r_2 + F_{S2} r_3 = 0$$

$$F_{S2} = m \ddot{s} + mg \quad F_{S1} = \frac{M_A}{r_1} - J_1 \frac{r_2}{r_1^2} \frac{\ddot{s}}{r_3}$$

$$(J_2 + J_3) \frac{\ddot{s}}{r_3} - \frac{r_2}{r_1} M_A + J_1 \frac{r_2^2}{r_1^2} \frac{\ddot{s}}{r_3} + m \ddot{s} r_3 + m g r_3 = 0$$

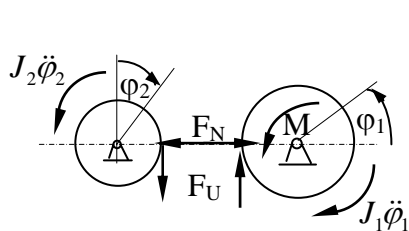
( $F_S = 0$ , da Riemenabtriebsseite ohne Last)

$$\ddot{s} = \frac{\frac{r_2}{r_1 r_3} M_A - mg}{\left(\frac{r_2}{r_3}\right)^2 \frac{J_1}{r_1^2} + \frac{(J_2 + J_3)}{r_3^2} + m} \quad 1. M_{Amin} \text{ für } \ddot{s} = 0 \Rightarrow M_A = M_{Amin} = \frac{r_1 r_3}{r_2} mg$$

$$2. M_A = 2M_{Amin} \Rightarrow \ddot{s} = \frac{mg}{\left(\frac{r_2}{r_3}\right)^2 \frac{J_1}{r_1^2} + \frac{(J_2 + J_3)}{r_3^2} + m} = K \quad \dot{s} = Kt \quad s = \frac{1}{2} Kt^2$$

$$3. t = \frac{\dot{s}}{K} \Rightarrow s = \frac{1}{2} \frac{\dot{s}^2}{K} \text{ oder } \ddot{s} = K = \frac{d\dot{s}}{ds} \dot{s} \quad \dot{s} d\dot{s} = K ds \quad \dot{s} = \sqrt{2Ks}$$

### Lösung 3.39



$$r_1 \dot{\phi}_1 = r_2 \dot{\phi}_2 \Rightarrow \dot{\phi}_1 = \frac{r_2}{r_1} \dot{\phi}_2$$

$$J_1 \ddot{\phi}_1 + F_U r_1 - M = 0$$

$$J_2 \ddot{\phi}_2 - F_U r_2 = 0 \Rightarrow F_U = \frac{J_2}{r_2} \ddot{\phi}_2$$

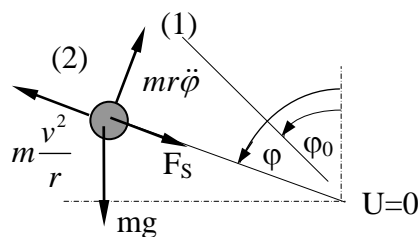
$$\ddot{\phi}_2 \left( J_1 \frac{r_2}{r_1} + J_2 \frac{r_1}{r_2} \right) - M = 0 \quad \ddot{\phi}_2 = \frac{M}{\left( J_1 \frac{r_2}{r_1} + J_2 \frac{r_1}{r_2} \right)} = K$$

$$\ddot{\phi}_2 = K \quad \dot{\phi}_2 = Kt + C \quad \dot{\phi}_2(0) = 0 \Rightarrow C = 0 \quad \dot{\phi}_2 = Kt$$

$$t = t_1: \quad \dot{\phi}_2 = \omega_2 \quad \omega_2 = Kt_1 \Rightarrow M = \frac{\omega_2}{t_1} \left( J_1 \frac{r_2}{r_1} + J_2 \frac{r_1}{r_2} \right) \quad \omega_2 = 2\pi n_2$$

$$J_1 = \frac{m_1}{2} r_1^2 = \frac{r_1^4}{2} \pi \rho b_1 \quad J_2 = \frac{m_2}{2} r_2^2 = \frac{r_2^4}{2} \pi \rho b_2 \quad M = \frac{\pi^2 n_2 \rho r_1 r_2}{t_1} (r_1^2 b_1 + r_2^2 b_2) = 4,6 Nm$$

### Lösung 4.2



$$U_1 + T_1 = U_2 + T_2$$

$$U_1 = mgr \cos \varphi_0 \quad T_1 = 0$$

$$U_2 = mgr \cos \varphi \quad T_2 = \frac{1}{2} mv^2(\varphi)$$

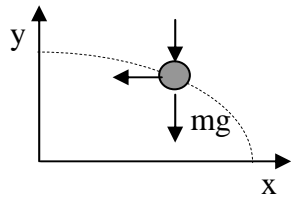
$$mgr \cos \varphi_0 = \frac{1}{2} mv^2(\varphi) + mgr \cos \varphi$$

$$v(\varphi) = \sqrt{2gr(\cos \varphi_0 - \cos \varphi)}$$

$$v(\pi) = \sqrt{2gr(\cos \varphi_0 + 1)} = 6,05 \frac{m}{s}$$

$$F_s(\varphi) = \frac{m}{r} v^2(\varphi) - mg \cos \varphi = mg(2 \cos \varphi_0 - 3 \cos \varphi) \quad F_s(\pi) = mg(\sqrt{3} + 3) = 928,4 N$$

### Lösung 4.10



Energiebilanz im Rohr:  $\frac{1}{2}cl^2 = \frac{1}{2}mv_0^2 \Rightarrow v_0 = l\sqrt{\frac{c}{m}}$

$$\ddot{x} = 0 \quad \dot{x} = C_1 \quad x = C_1t + C_2$$

$$\ddot{y} = -g \quad \dot{y} = -gt + C_3 \quad y = -\frac{1}{2}gt^2 + C_3t + C_4$$

$$AB: t=0 \quad x=0 \Rightarrow C_2=0 \quad \dot{x}=v_0 \Rightarrow C_1=v_0$$

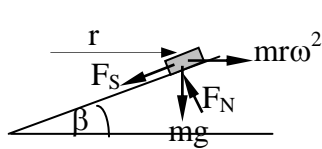
$$y=h \Rightarrow C_4=h \quad \dot{y}=0 \Rightarrow C_3=0$$

$$EB: t=t_E \quad x=w \quad y=0 \Rightarrow w=v_0t_E \quad t_E = \frac{w}{v_0} \quad \frac{1}{2}g\left(\frac{w}{v_0}\right)^2 - h = 0 \quad l = \sqrt{\frac{w^2mg}{2ch}}$$

Zahlenwerte:  $l = 9,9 \text{ cm}; \quad v_0 = 8,09 \text{ m/s}$

### Lösung 4.13

Bewegung auf der schiefen Ebene:  $mgl \sin \alpha = \frac{1}{2}mv_1^2 \Rightarrow v_1 = \sqrt{2gl \sin \alpha}$



$$\dot{\varphi} = \omega = \frac{v_1}{r}$$

$$\rightarrow: mr\omega^2 - F_S \cos \beta - F_N \sin \beta = 0$$

$$\downarrow: mg + F_S \sin \beta - F_N \cos \beta = 0 \quad F_N = \frac{mg + F_S \sin \beta}{\cos \beta}$$

$$mr\omega^2 - F_S \cos \beta - \frac{mg + F_S \sin \beta}{\cos \beta} \cdot \sin \beta = 0$$

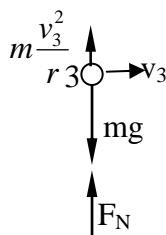
$$F_S = 0 \quad \text{für} \quad \beta = \beta^* \quad \tan \beta^* = \frac{r\omega^2}{g} \quad \text{Überhöhung: } h = w \sin \beta^* = w \frac{\tan \beta^*}{\sqrt{1 + \tan^2 \beta^*}}$$

$$h = \frac{w}{\sqrt{1 + \left(\frac{r}{2l \sin \alpha}\right)^2}} = 20,6 \text{ cm}$$

### Lösung 4.14

$$T_1 + U_1 = T_2 + U_2 \Rightarrow mgh_1 + \frac{1}{2}mv_1^2 = \frac{1}{2}mv_2^2 \quad v_2 = \sqrt{2gh_1 + v_1^2}$$

$$T_1 + U_1 = T_3 + U_3 \Rightarrow mgh_1 + \frac{1}{2}mv_1^2 = mgh_3 + \frac{1}{2}mv_3^2 \quad v_3 = \sqrt{2g(h_1 - h_3) + v_1^2}$$



$$\uparrow: F_N - mg + m \frac{v_3^2}{r} = 0 \quad F_N = mg - m \frac{v_3^2}{r}$$

$$F_N = 0 \quad \text{für} \quad r = r^* \quad \Rightarrow \quad r^* = \frac{v_3^2}{g}$$

$$r > r^* = 2(h_1 - h_3) + \frac{v_1^2}{g} \quad v_2 = 6,95 \frac{m}{s} \quad r > 2,917m$$